

assembled by ultrasonic welding or laser welding, the production cost may be decreased.

[0081] Hereinafter, an air conditioning apparatus to which a turbofan according to an embodiment of the present disclosure is applied will be described with reference to FIGS. 11 and 12.

[0082] FIG. 11 is a perspective view illustrating a ceiling air conditioning apparatus in which a turbofan for an air conditioning apparatus according to an embodiment of the present disclosure is used. FIG. 12 is a cross-sectional view illustrating the ceiling air conditioning apparatus of FIG. 11 taken along a line 12-12.

[0083] For reference, the air conditioning apparatus 300 as illustrated in FIGS. 11 and 12 is a ceiling air conditioning apparatus; however, the present disclosure is not limited thereto. The turbofan 1 according to an embodiment of the present disclosure may be used in various types of air conditioning apparatuses.

[0084] Referring to FIGS. 11 and 12, the air conditioning apparatus 300 may include a housing 310, a heat exchanger 320, and a turbofan 1.

[0085] The housing 310 is formed in a substantially hollow rectangular parallelepiped, forms an appearance of the air conditioning apparatus 300, and is provided in a ceiling 350. The bottom surface of the housing 310 is provided with an air inlet opening 311 which is exposed to the outside of the ceiling 350 and faces the suction port 23 of the turbofan 1. Also, the bottom surface of the housing 310 is provided with a plurality of air discharge ports 312 around the air inlet opening 311. A filter to filter incoming air may be disposed below the air inlet opening 311.

[0086] The turbofan 1 is disposed in the center of the housing 310, and is rotated by a motor 330 provided in the upper side of the housing 310. The turbofan 1 includes the shroud 20, the hub plate 10, and the plurality of blades 30. The structure of the turbofan 1 is the same as or similar to the structure of the turbofan 1 according to the above-described embodiment; therefore, a detailed description thereof will be omitted.

[0087] A bell mouth 340 for stabilizing the incoming air is provided between the shroud 20 of the turbofan 1 and the bottom surface of the housing 310 in which the air inlet opening 311 is formed. Also, a gap may be provided between an end 341 of the bell mouth 340 and portions of the plurality of blades 30 of the turbofan 1 facing the bell mouth 340.

[0088] The heat exchanger 320 is provided so as to surround the turbofan 1 inside the housing 310. The heat exchanger 320 cools the air by exchanging heat with the air passing through the heat exchanger 320.

[0089] When the motor 330 rotates the turbofan 1, a negative pressure is generated in the inside of the turbofan 1 so that the outside air flows into the suction port 23 of the turbofan 1 through the filter 314. The air flowing into the suction port 23 of the turbofan 1 is discharged to the heat exchanger 320 provided around the turbofan 1 by the plurality of blades 30. The air discharged by the turbofan 1 is cooled while passing by the heat exchanger 320. The cooled air is discharged through the air discharge ports 312 of the housing 310, thereby cooling a room in which the air conditioning apparatus 300 is disposed.

[0090] Hereinafter, improved performance of the air conditioning apparatus that uses a turbofan according to an

embodiment of the present disclosure having a structure as described above will be described in detail with reference to FIGS. 13A to 18.

[0091] Performance of the air conditioning apparatus 300 receives much influence by the distribution of the air flow being formed by the turbofan 1.

[0092] FIGS. 13A and 13B are views illustrating a flow of air in a blade of a conventional two-dimensional turbofan. FIGS. 14A and 14B are views illustrating a flow of air in a blade of a turbofan for an air conditioning apparatus according to an embodiment of the present disclosure. FIGS. 13A to 14B show results obtained by computer simulation.

[0093] Referring to FIG. 13A, it can be seen that a lot of turbulence 150 occur at a separation point of the entrance portion 131 of the blade 130 of the conventional two-dimensional turbofan 100. In the exit portion 132 of the blade 130, due to the influence of the turbulence 150, as illustrated in FIG. 13B, the flow field of air is distributed in approximately half the height of the blade 130.

[0094] However, in the turbofan 1 for an air conditioning apparatus according to an embodiment of the present disclosure, as illustrated in FIG. 14A, compared to the conventional two-dimensional turbofan 100, less turbulence 50 occurs at the separation point of the entrance portion 31 of the blade 30 by the effect of the curved portion 31-1. In the turbofan 1 for an air conditioning apparatus according to an embodiment of the present disclosure, since the turbulence 50 occurs less, the flow field of air is distributed over substantially the entire height of the blade 30 in the exit portion 32 of the blade 30 as illustrated in FIG. 14B.

[0095] The results of measuring power consumption and noise of the turbofan 1 according to an embodiment of the present disclosure and the conventional two-dimensional turbofan are shown in FIGS. 15 and 16.

[0096] FIG. 15 is a graph comparing power consumption of a turbofan for an air conditioning apparatus according to an embodiment of the present disclosure and the conventional two-dimensional turbofan. FIG. 16 is a graph comparing noise of a turbofan for an air conditioning apparatus according to an embodiment of the present disclosure and the conventional two-dimensional turbofan.

[0097] In FIGS. 15 and 16, measurement data of the turbofan 1 according to an embodiment of the present disclosure are measured after replacing the two-dimensional turbofan 100 of the air conditioning apparatus, which is provided with the two-dimensional turbofan 100 and is used in the experiment, with the turbofan 1 according to an embodiment of the present disclosure.

[0098] Referring to FIG. 15, it can be seen that the power consumption of the turbofan 1 according to an embodiment of the present disclosure is decreased about 11% at the same flow rate as compared to the conventional two-dimensional turbofan. In FIG. 15, P represents the power consumption (unit; W) of the turbofan, and Q represents the flow rate (unit; CMM) of the turbofan.

[0099] Referring to FIG. 16, it can be seen that the noise of the turbofan 1 according to an embodiment of the present disclosure is decreased about 0.5 dB at the same flow rate as compared to the conventional two-dimensional turbofan. In FIG. 16, SPL (sound pressure level) represents the noise (unit; dB) generated by the turbofan, and Q represents the flow rate (unit; CMM) of the turbofan.

[0100] As described above, in the turbofan 1 according to an embodiment of the present disclosure, the power con-